

Performance of Ammonia/Natural Gas Co-Fired Gas Turbine with Two-Stage Combustor

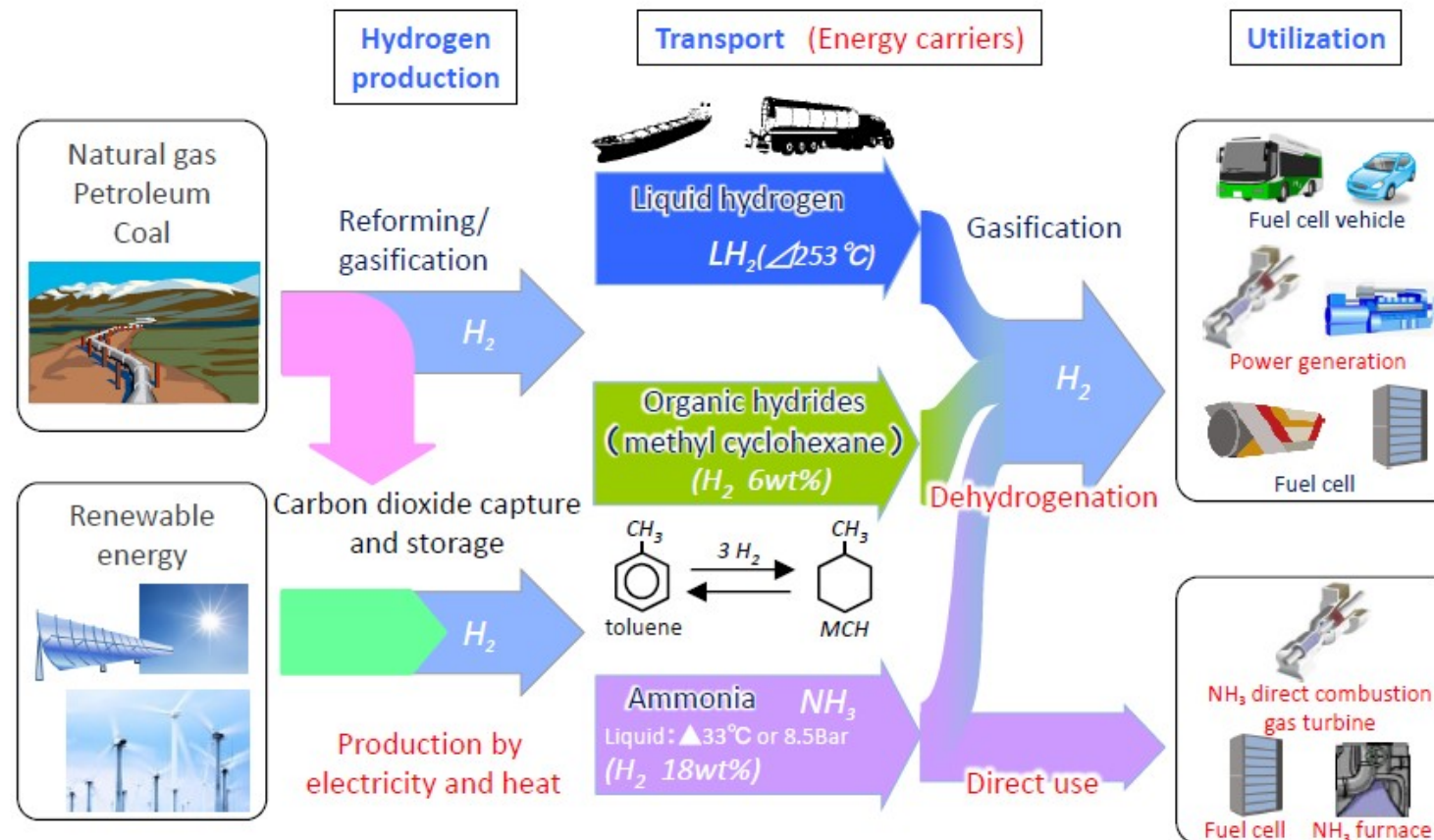


Masahiro Uchida, Shintaro Ito,
Toshiyuki Suda, Toshiro Fujimori

IHI Corporation

Advantages of ammonia as an energy carrier

- (1) Highest hydrogen content per unit volume
- (2) Easy to liquefy (-33°C at 1bar, similar to LPG)
- (3) Infrastructures for production and transportation are already existing
- (4) Can be used directly as a fuel for power plant



Energy carriers considered in 'SIP Energy Carriers' project

Problems to overcome

- (1) **Optimized combustor design for stable flame and reduction of fuel-NOx**
- (2) Evaluation of performance of the power plant
- (3) Safety measures
- (4) Cost (Feasibility studies)



IHI joined Cross-ministerial Strategic Innovation Promotion Program (SIP) for the development of **Ammonia Direct Combustion** technology for gas turbine and **coal fired boiler** and also **Ammonia Fuel Cell**.



戦略的イノベーション創造プログラム
Cross-ministerial Strategic Innovation Promotion Program

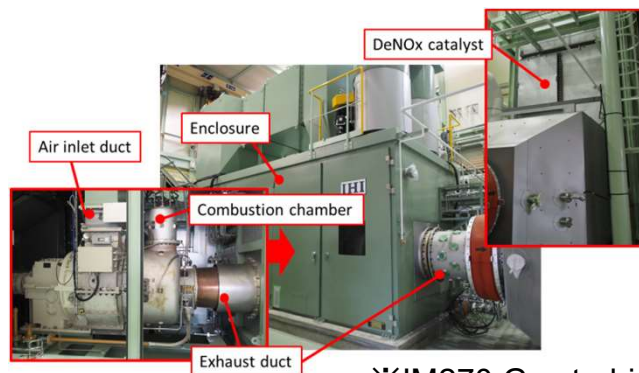


Coal fired boiler



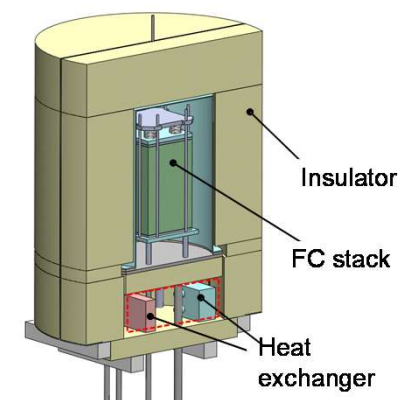
※CFT(Coal Firing Test Furnace)

Gas turbine



※IM270 Gas turbine

SOFC

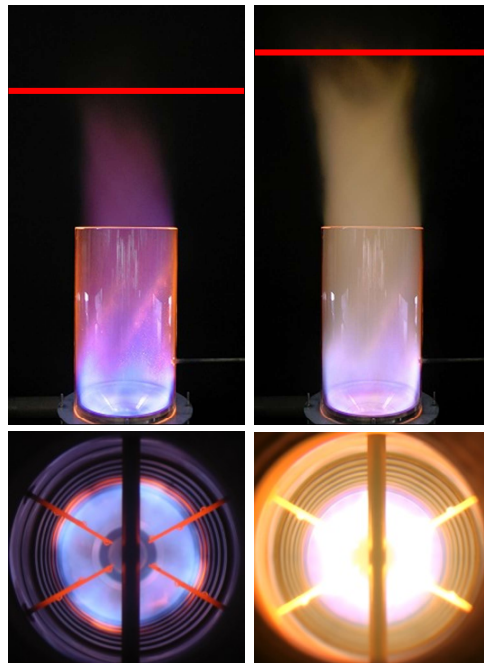


Target power plant of 'SIP Energy Carriers' project in IHI

Task :

Optimization of combustor design to reduce NOx

Demonstration using 2MW scale commercial gas turbine



City gas

Ammonia
co-firing

Comparison of swirl flame

Feature of NH_3 combustion (compared with CH_4)

- Low flame speed (approx. 1/5)
- Low heating value (approx. 1/2)
- Low flame temperature (approx. 200°C)
- Emission of fuel-NOx (Thermal NOx)

Problems to be solved

- Burner design to achieve stable flame
- Reduction of fuel-NOx
- Reduction of unburned NH_3

Item	Value
Engine	IM270 manufactured by IHI Corporation
Power generation output	2MWe
Cycle	Simple cycle
NH ₃ mixing ratio	Maximum 20-25%LHV

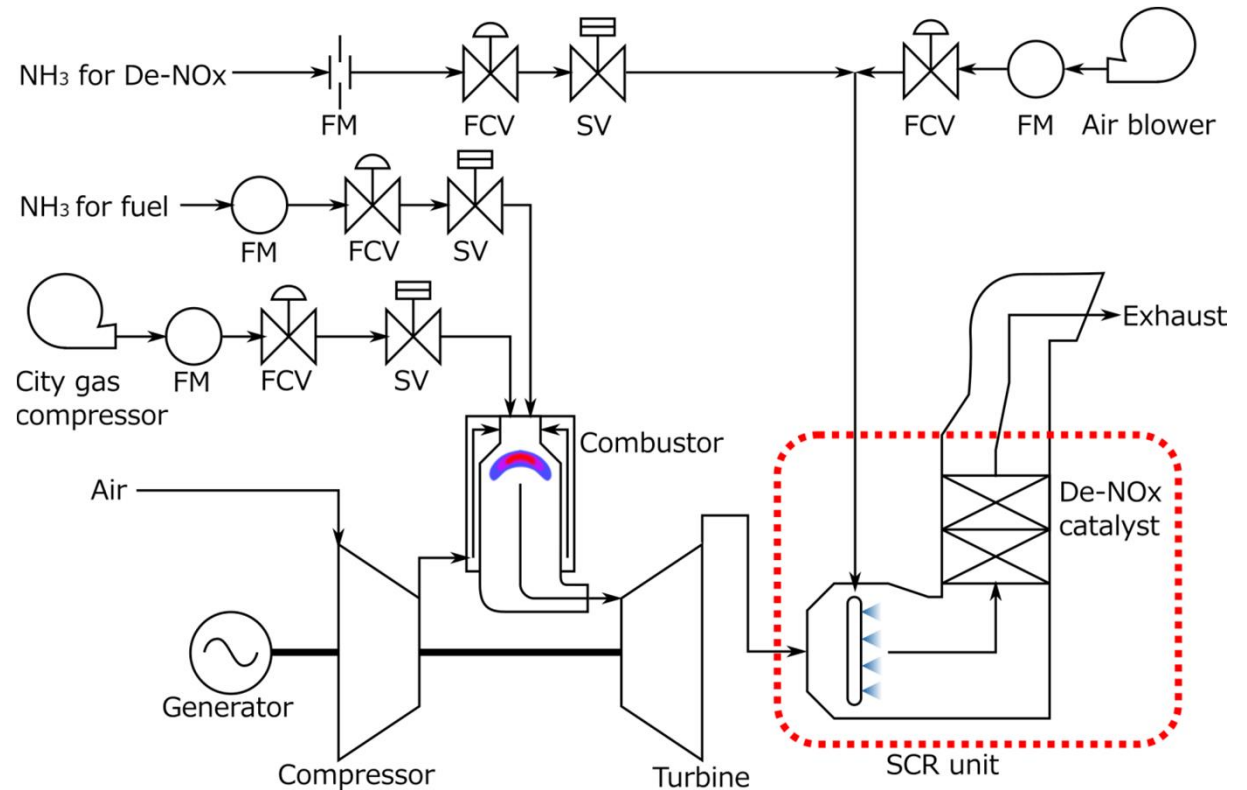
NH₃ mixing ratio (based on LHV)

$$r_{NH_3} [\%LHV] = 100 \times \frac{\text{Energy input of NH}_3 [\text{kW}]}{\text{Energy input of Natural gas} [\text{kW}] + \text{Energy input of NH}_3 [\text{kW}]}$$

Demonstration using commercial 2MW class GT

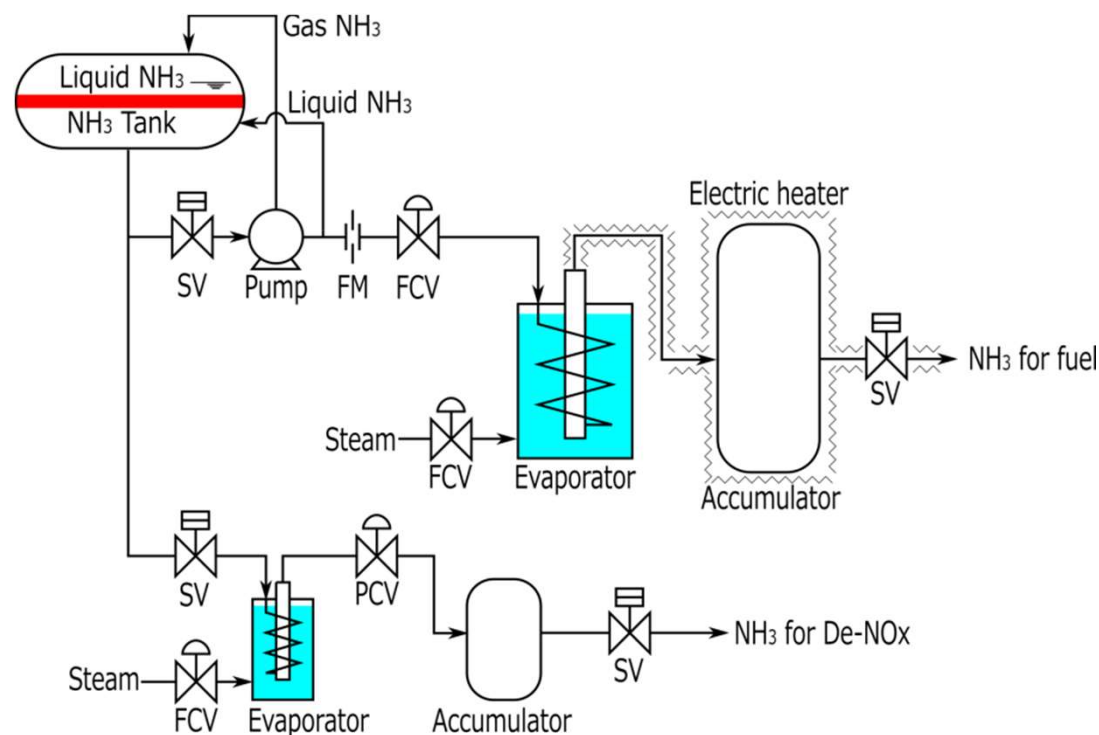
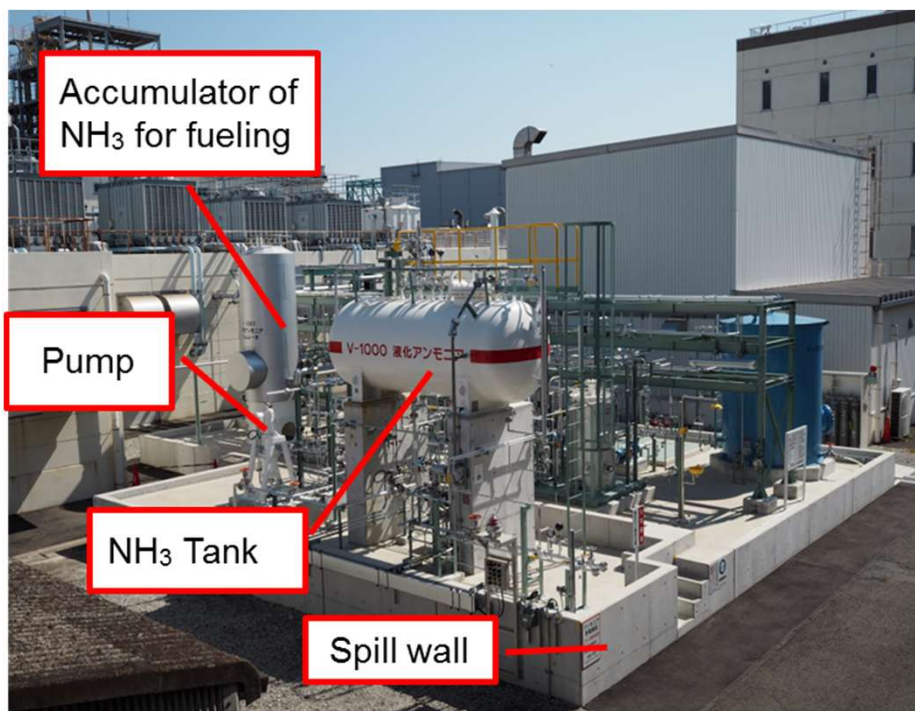


IM270 gas turbine



- Only the combustor was modified to co-fire NH₃ / Natural gas
- Selective Catalytic Reduction (SCR) unit for NO_x reduction in exhaust gas
- Pressurized gasified NH₃ provided by NH₃ supply system

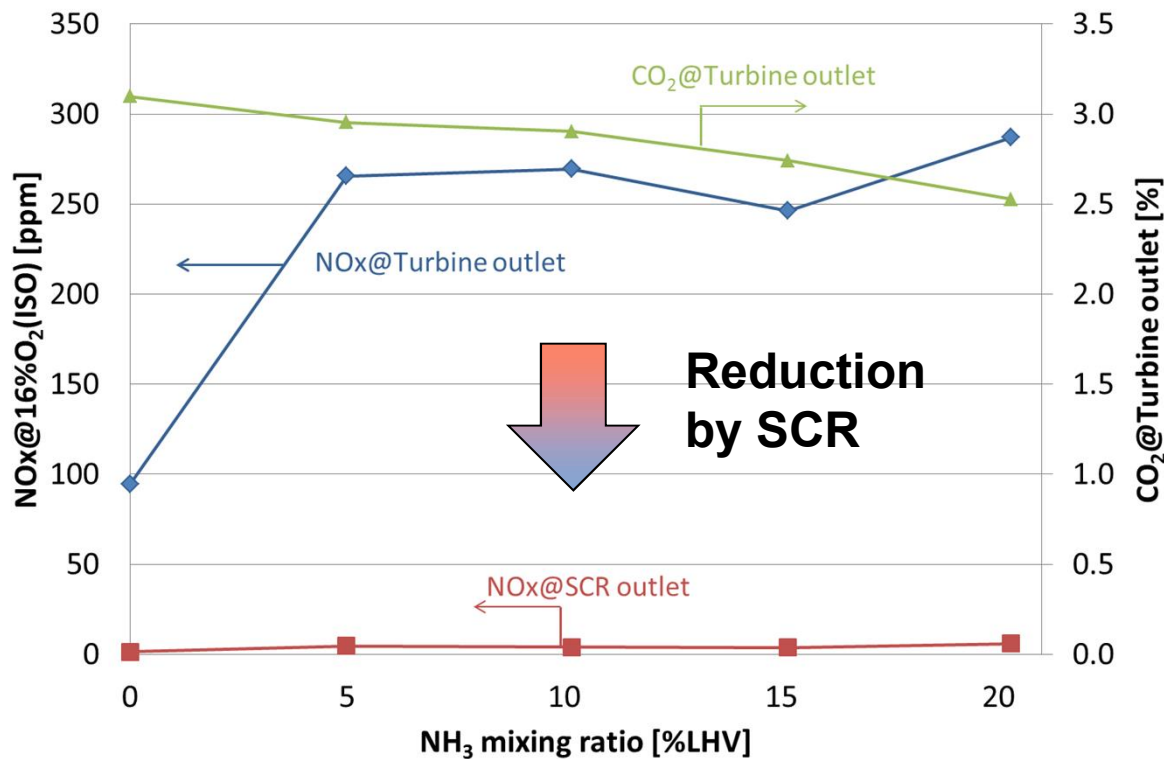
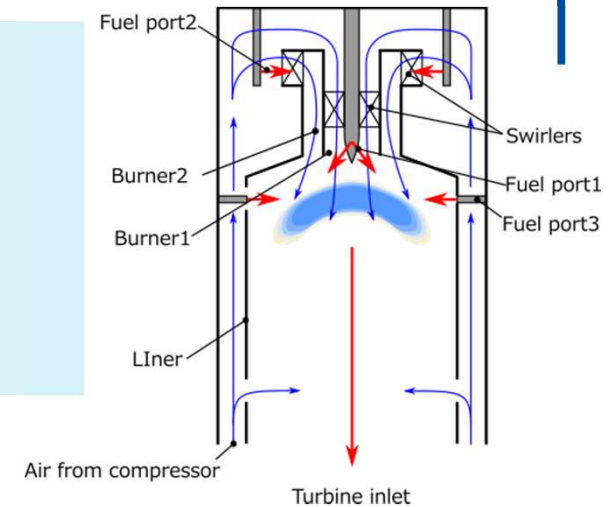
High pressure, gasified NH₃ Supply System



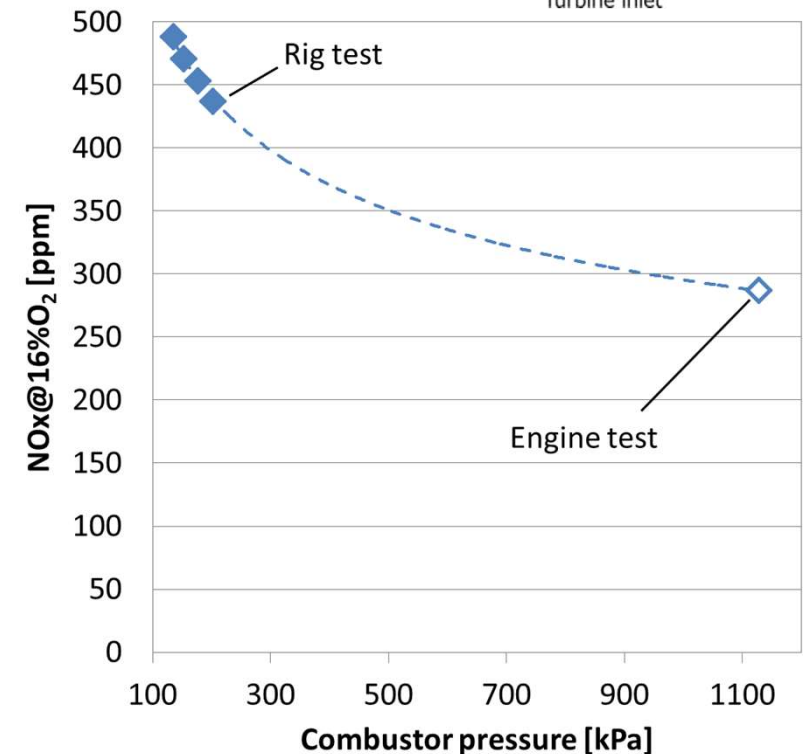
- Liquid NH₃ is pressurized by pump, then vaporized by hot water bath evaporator
- Feed lines and accumulator is heated to prevent re-liquefaction of gasified NH₃
- NH₃ for NO_x reduction at SCR unit fed from separate low-pressure supply line

Previous results with Lean Premixed Combustor

- Results with Lean Premixed Combustor with Secondary Port
- NOx emission increases with increasing NH3 mixing ratio
290ppm @ NH3 mixing ratio = 20%
- NOx emission decreases with increasing test pressure



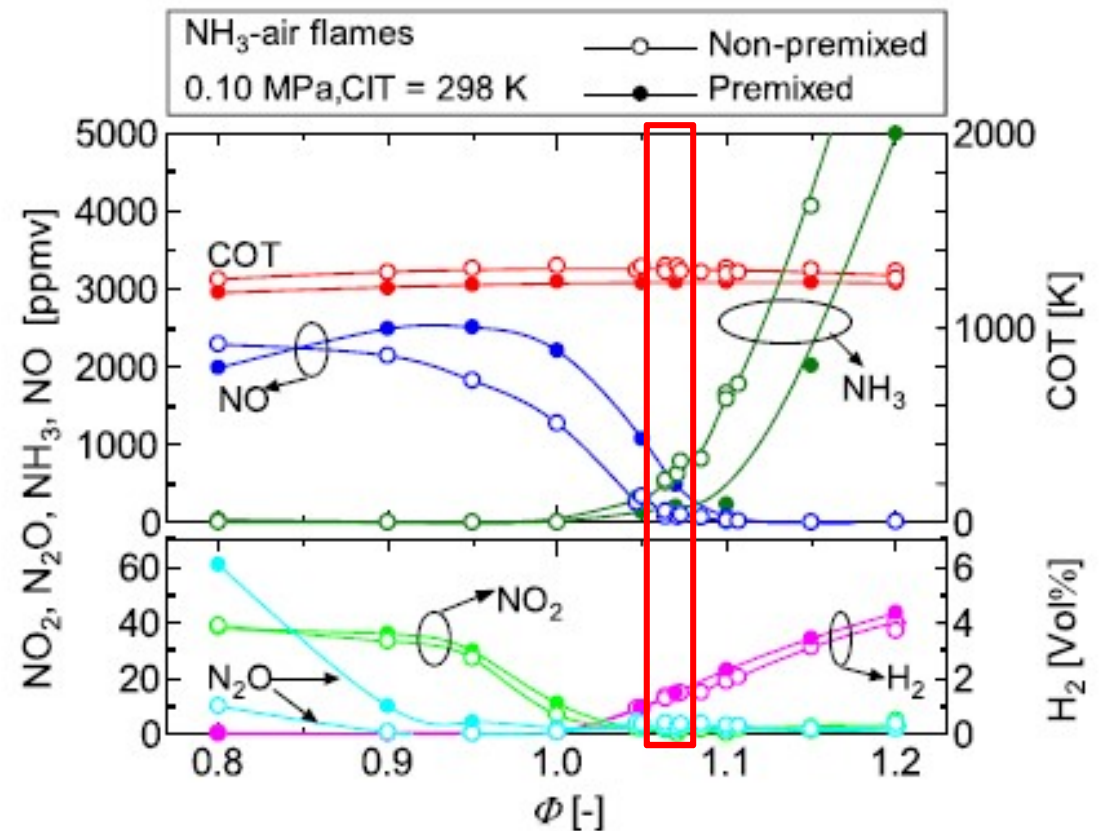
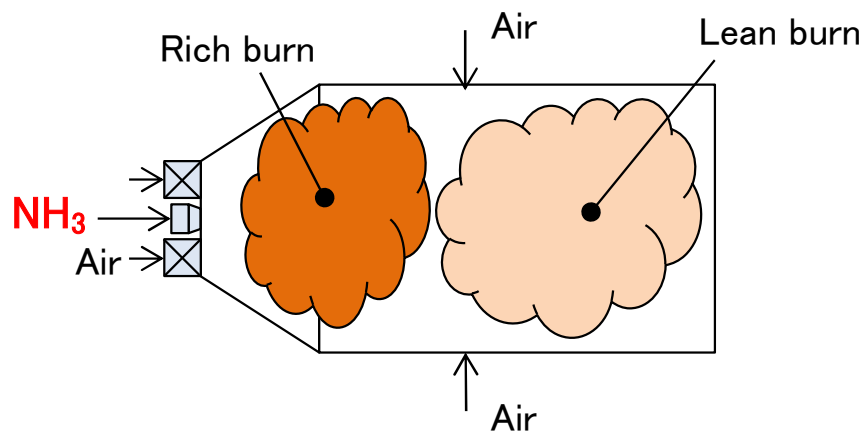
Effect of co-firing ratio on NOx emission



Effect of pressure on NOx emission

Low NOx combustion with Two-Stage Combustor

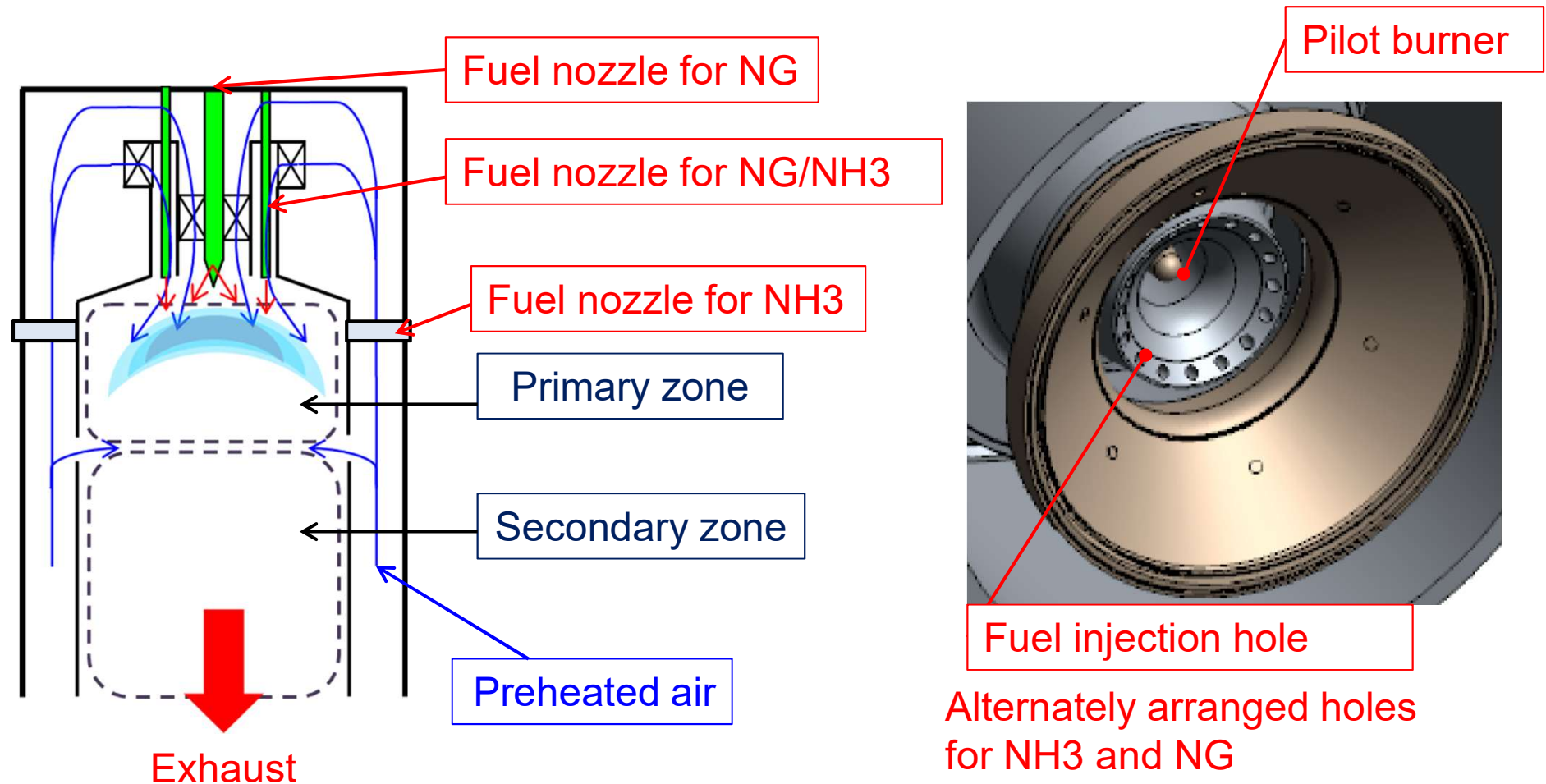
- To further decrease NOx emission, Rich-lean (or Two-stage) combustor is effective
Hayakawa et al.(2017), Kurata et al.(2017), Okafor et al.(2019)
- Minimum NOx condition around equivalence ratio of 1.07 for pure NH₃ combustion
=> Examine the effect of two-stage combustor in rig and engine tests



Effect of primary zone equivalence ratio on emission
(Hayakawa et al., J. Hydrogen Energy, 2017)

Two-stage Combustor with Secondary Burner

- Risk of flashback \Rightarrow diffusion flame burner
- Two injection port for NH_3 : ① Main fuel nozzles, ② Secondary port
- Primary equivalence ratio: 0.85, 1.1, 1.4



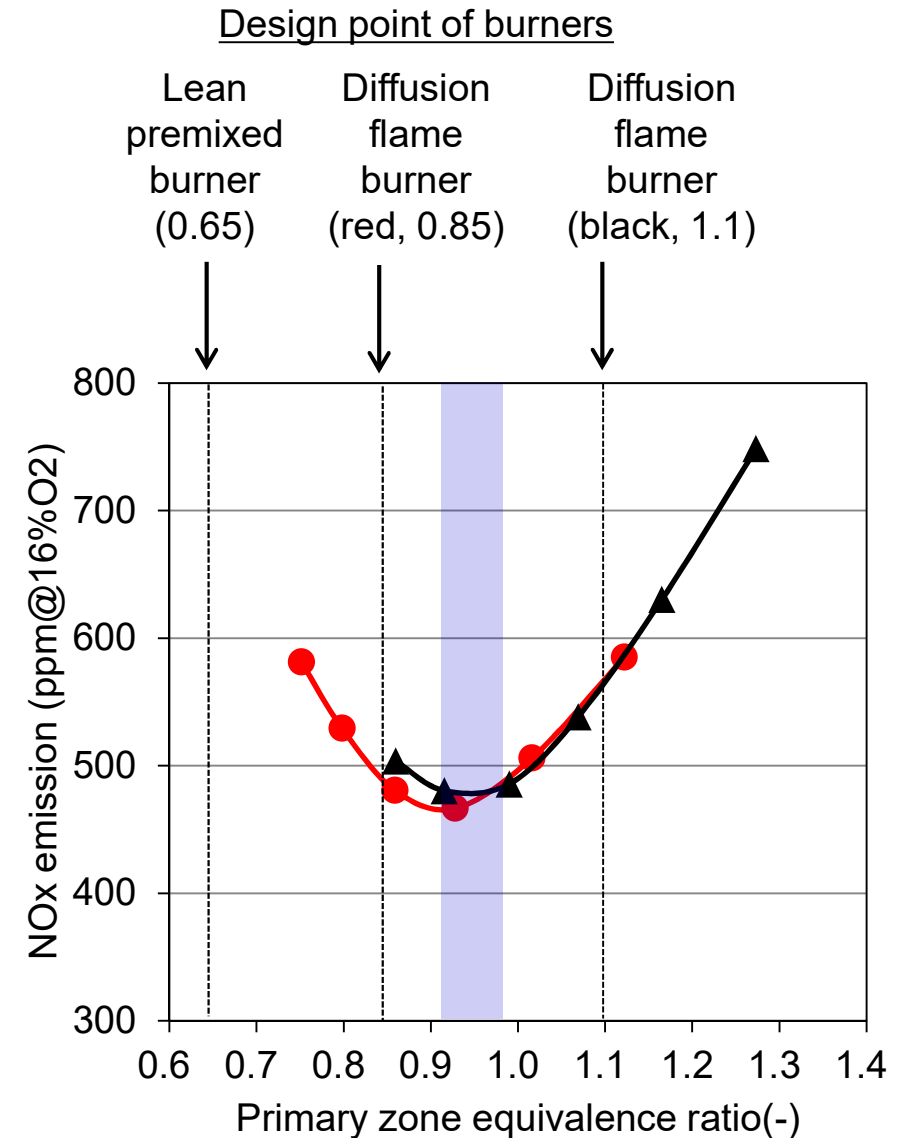
Results of rig tests (1)

Result of rig tests at atmospheric condition

NH3 mixing ratio = 20%

- Air flow rate was changed to vary primary zone equivalence ratio
- NOx is evaluated by converted into 16% O2 condition to elucidate the effect of dilution
- Minimum NOx around primary zone equivalence ratio of 0.95
- Same results for two burners
: effect of residence time is small
- Results not consistent with calculations
- Burner with primary equivalence ratio of 0.85 is expected to be most low NOx

NH3 injection from secondary burner



Results of rig tests (2)

Result of rig tests at atmospheric condition

NH₃ injection from main burner

NH₃ mixing ratio = 20%

- Minimum NO_x around primary zone equivalence ratio of 1.2
- NO_x emission is much higher than secondary NH₃ injection case



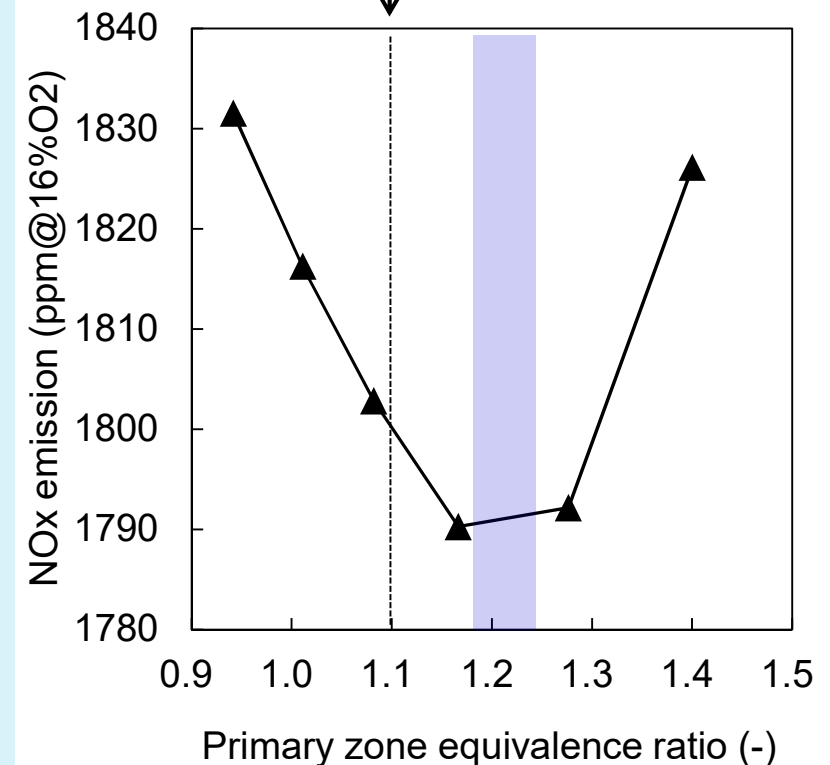
Main: NH₃-Air diffusion flame

Secondary: NH₃-Burned gas (Natural gas -Air)
diffusion flame

- Local progress of mixing and reactions might effects NO_x emissions
Direct contact of ammonia and air
Radical and intermediate species distributions

Design point of burners

Diffusion
flame
burner
(black, 1.1)

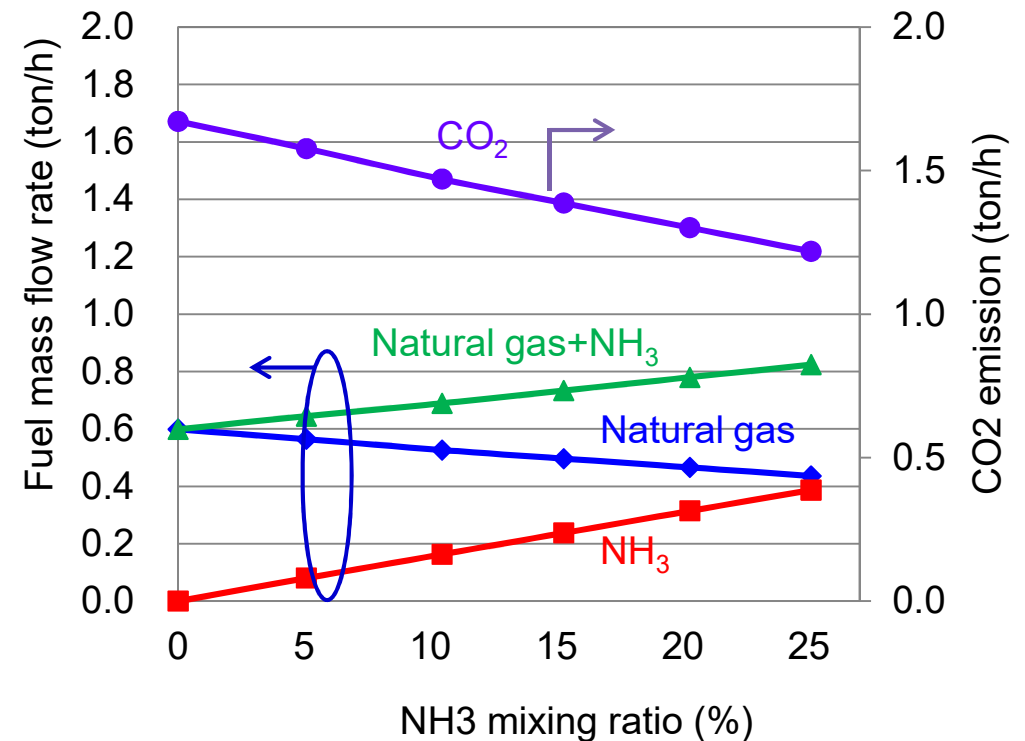
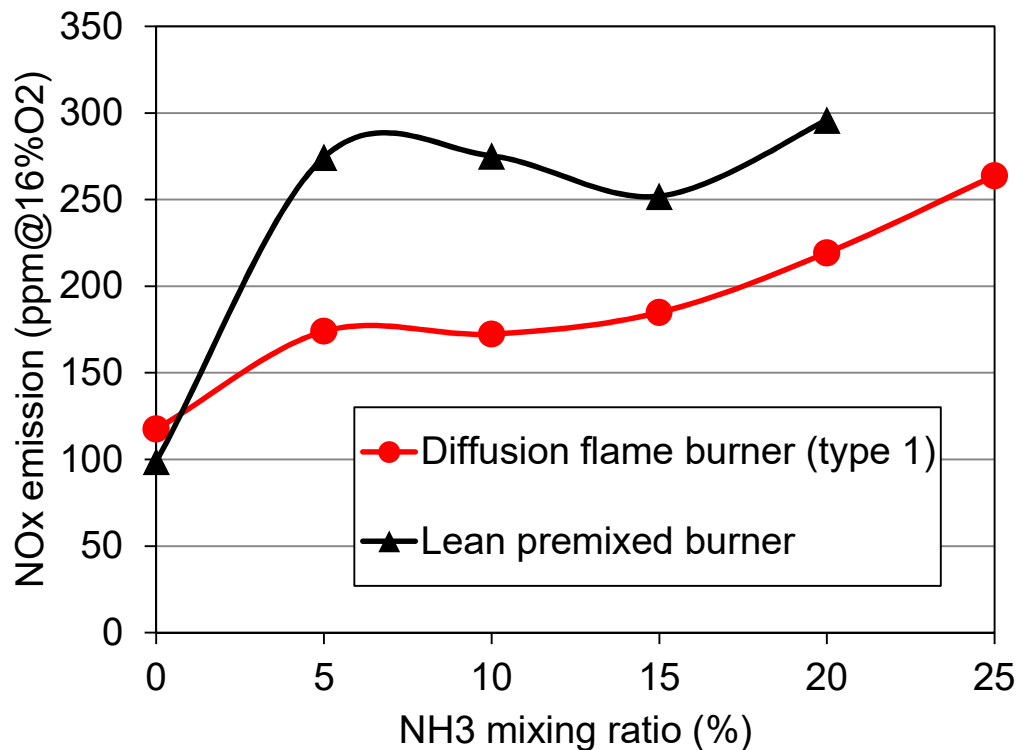


Results of engine test

Diffusion burner with primary equivalence ratio of 0.85

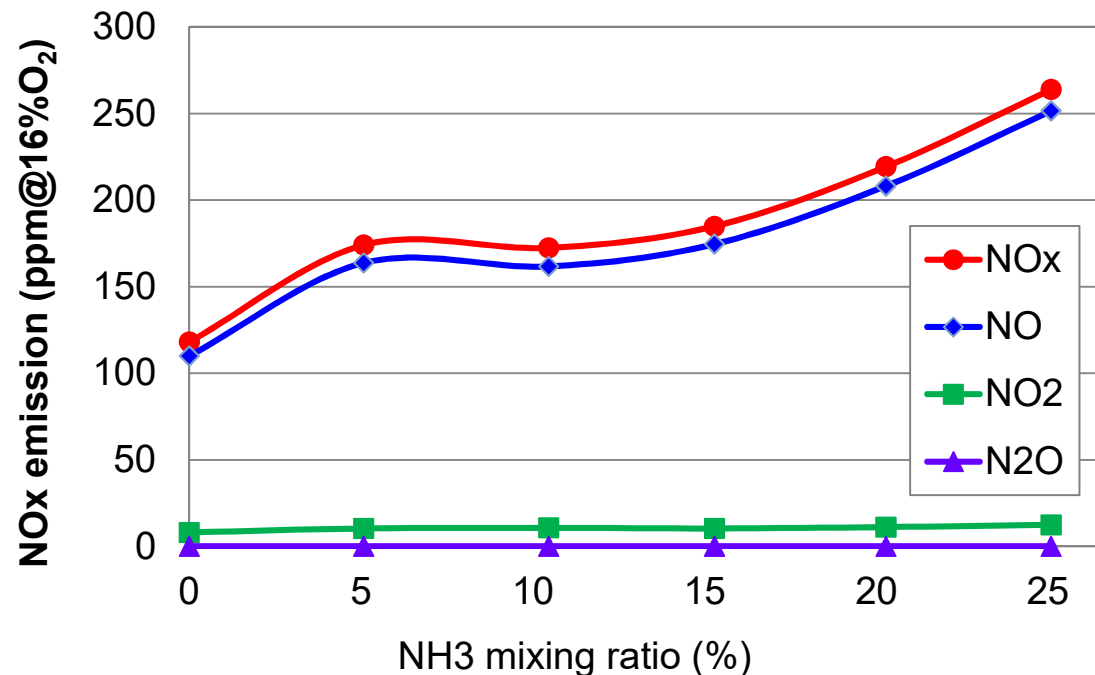
NH3 is injected from secondary burner

- NOx emission: 230ppm @ NH3 mixing ratio = 20%
- Reduction of CO₂ is confirmed in engine test



Composition of NOx emission in engine test

- Composition of NOx is also measured with QL-NX of Horiba LTD
QCL-IR (Quantum Cascade Laser – Infrared Spectroscopy) is adopted
- Reduction of N₂O is important due to its high GWP
GWP(Global Warming Potential) CO₂: 1, CH₄: 25, N₂O: 298
- **N₂O emission is suppressed in GT condition,**
though NOx emission is higher than NG combustion case
N₂O < 1ppm (under measurements range)



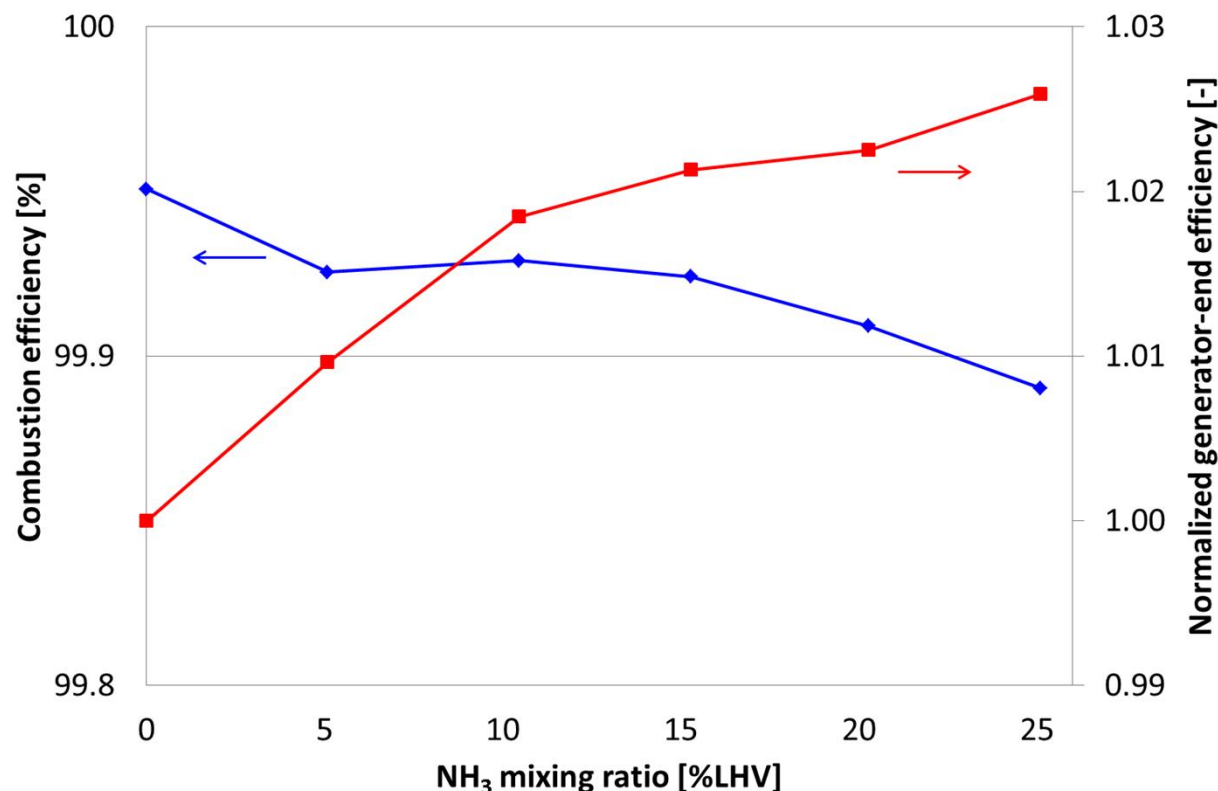
- Other unburned components are also suppressed

$\text{NH}_3 < 12 \text{ ppm}$, $\text{CO} < 4 \text{ ppm}$, Total Hydro Carbon $< 1 \text{ ppm}$

As a results, combustion efficiency of over 99.85% is achieved in all test conditions

- Generator-end efficiency slightly improved by increasing ammonia mixing ratio

This is because flow rate of burned gases are increased by increasing ammonia, due to low calorific value of ammonia than natural gas.



Effects of primary zone equivalence ratio and secondary NH₃ injections are investigated by rig testing and engine testing.

Results of rig testing show that,

- Optimal primary equivalence ratio depends on the ammonia injection method
- The lowest NO_x combustion is achieved with secondary injection case

Results of engine testing show that,

- NO_x emissions can be reduced by diffusion flame burner
- Two-stage combustion method is effective in reducing NO_x even for secondary ammonia injection method, but the mechanism might be slightly different

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