



# Pure Ammonia Combustion Micro Gas Turbine System

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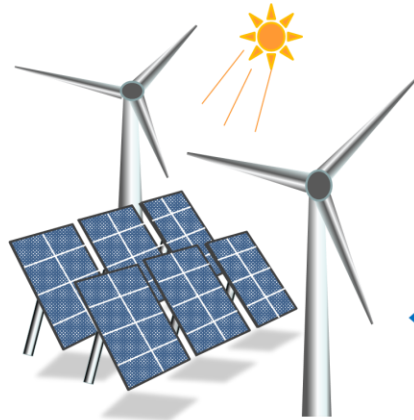
# Outline

- Why ammonia...
- Challenges of ammonia combustion
- The pure ammonia micro gas turbine
  - the achievements and challenges
- Fundamental studies
- Development of a low-NO<sub>x</sub> combustor

# 1. Why ammonia...



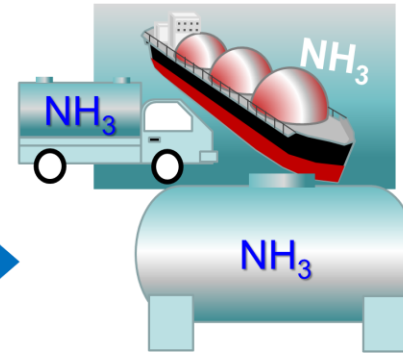
Has a high  $H_2$  density (17.6 %wt)



Can be produced from **renewable** energy sources.

Has long history of production through Haber-Bosch process.

Ammonia is a promising **carbon-free** fuel and a hydrogen carrier!



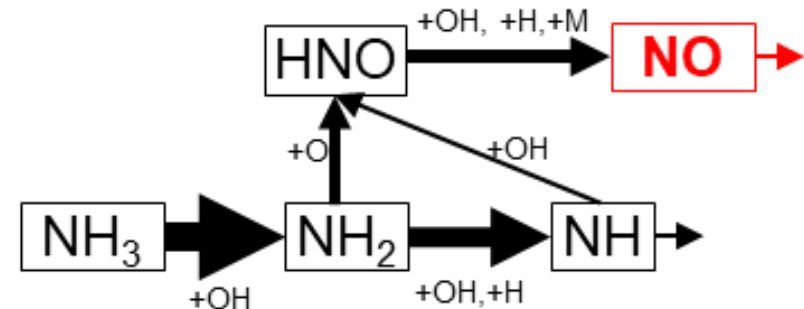
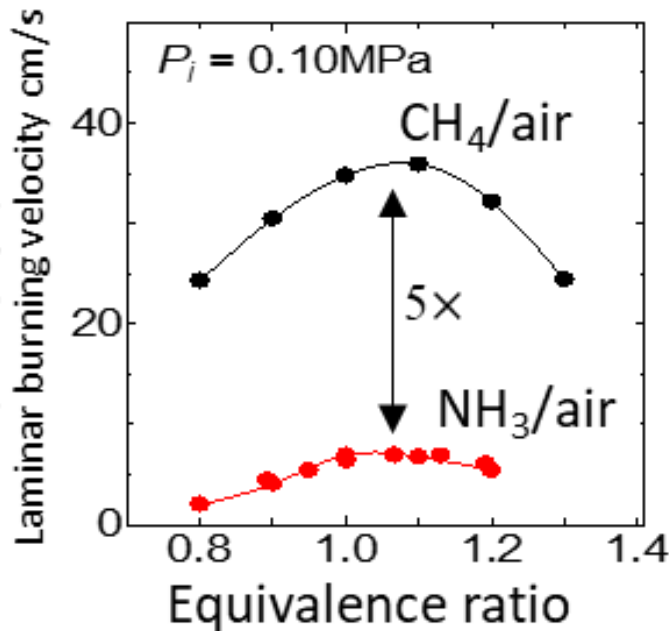
Is easier to store and transport than hydrogen  
- **liquid at 8.5 bar at 25 °C.**



Zero  $CO_2$   
Zero soot  
Zero  $SO_x$

Ammonia-air flame  
in a swirl  
combustor

Wide use of ammonia as a gas turbine fuel would ensure a huge reduction in global  $CO_2$  emissions.



## Low burning velocity

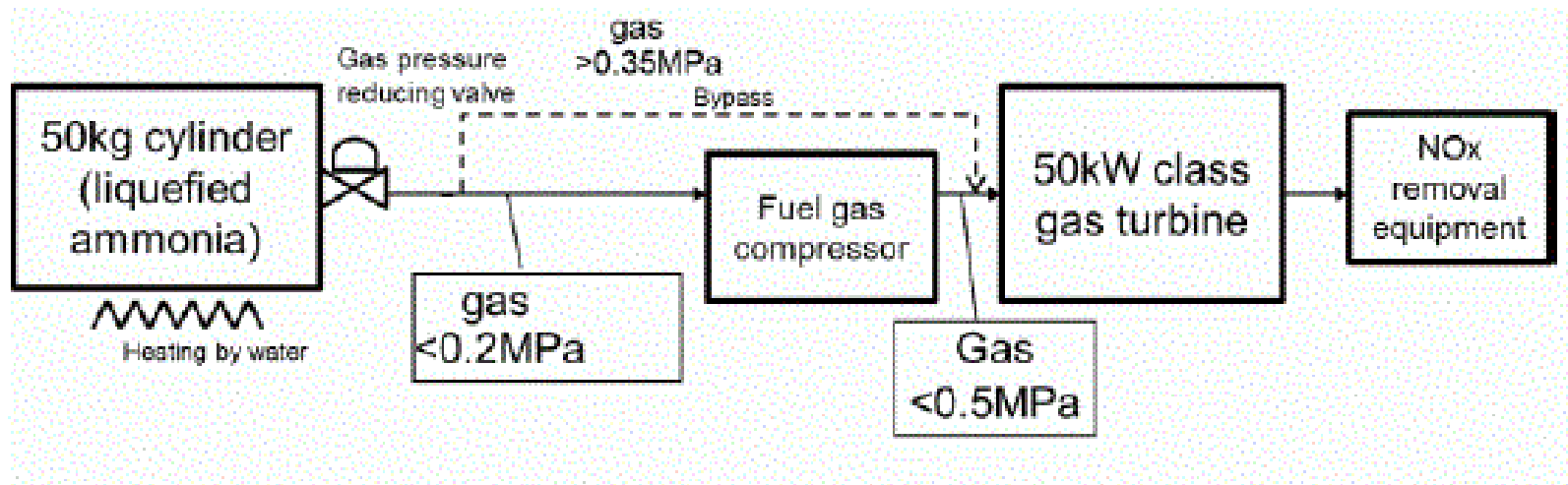
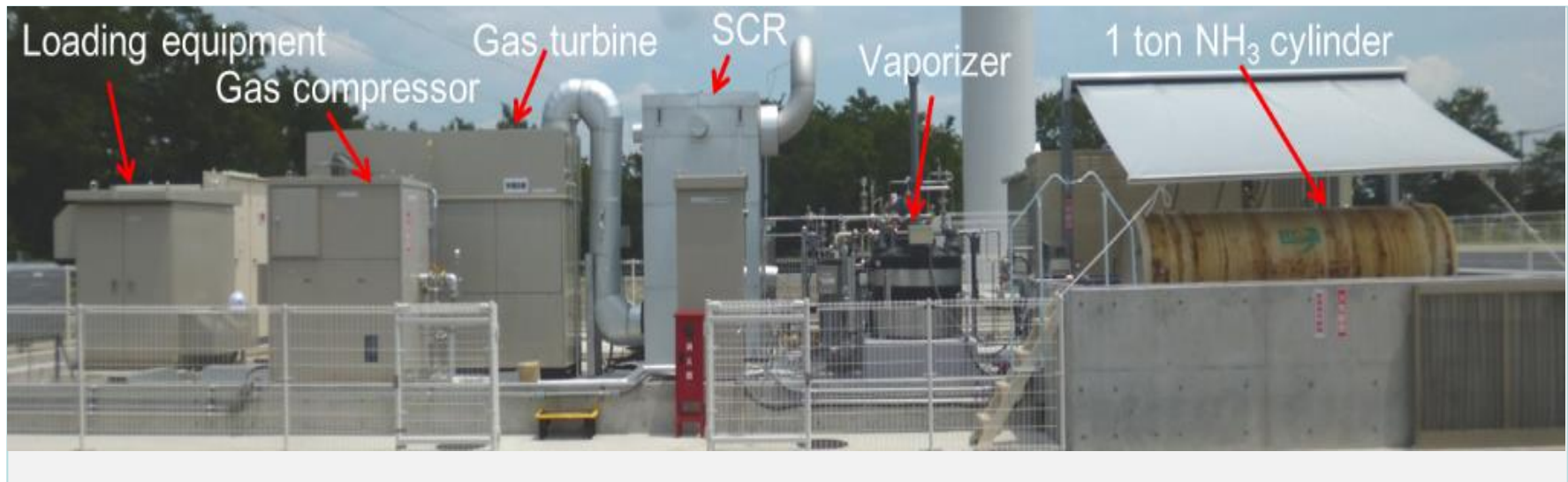
- Inhibits flame stabilization
- May lead to low combustion efficiencies in turbine combustors

## High NO emissions at fuel-lean conditions

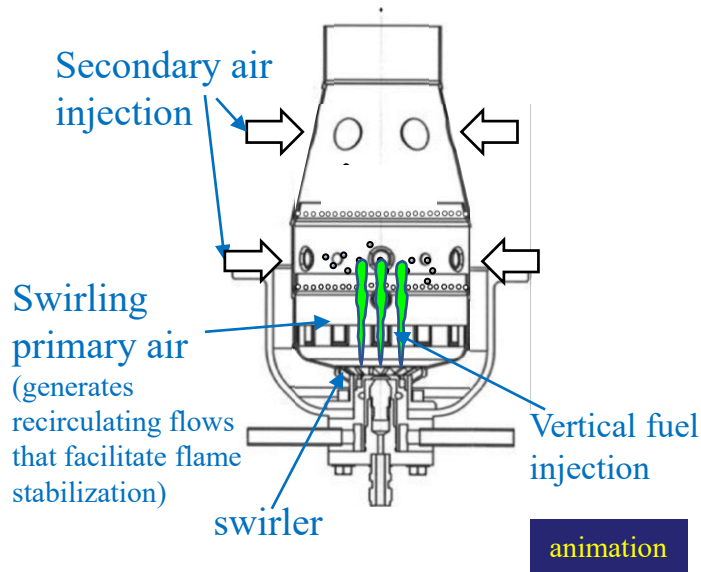
- Fuel-lean ammonia flames produce high NO<sub>x</sub> emissions.

**These challenges discouraged the application of ammonia as a fuel until recently.**

## Generation of 41.4 kW from pure ammonia in a micro gas turbine

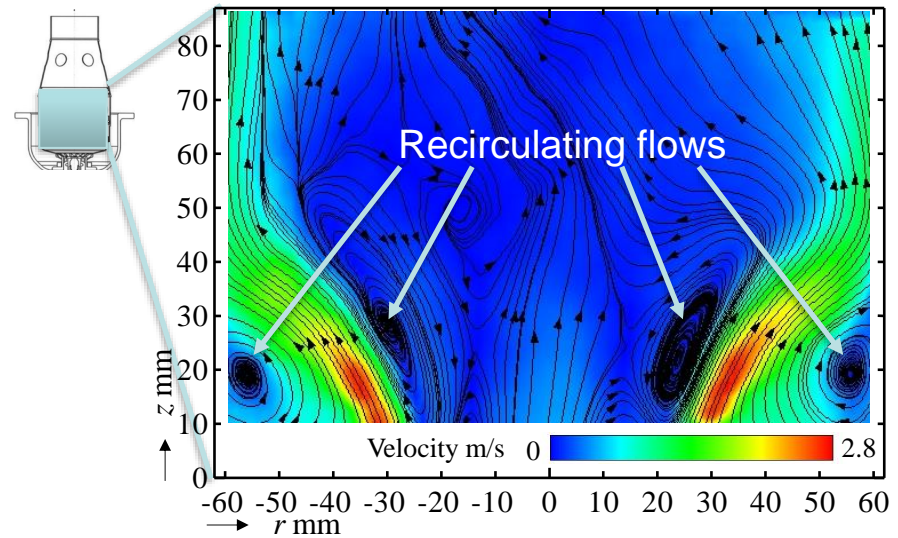


## THE ORIGINAL COMBUSTOR



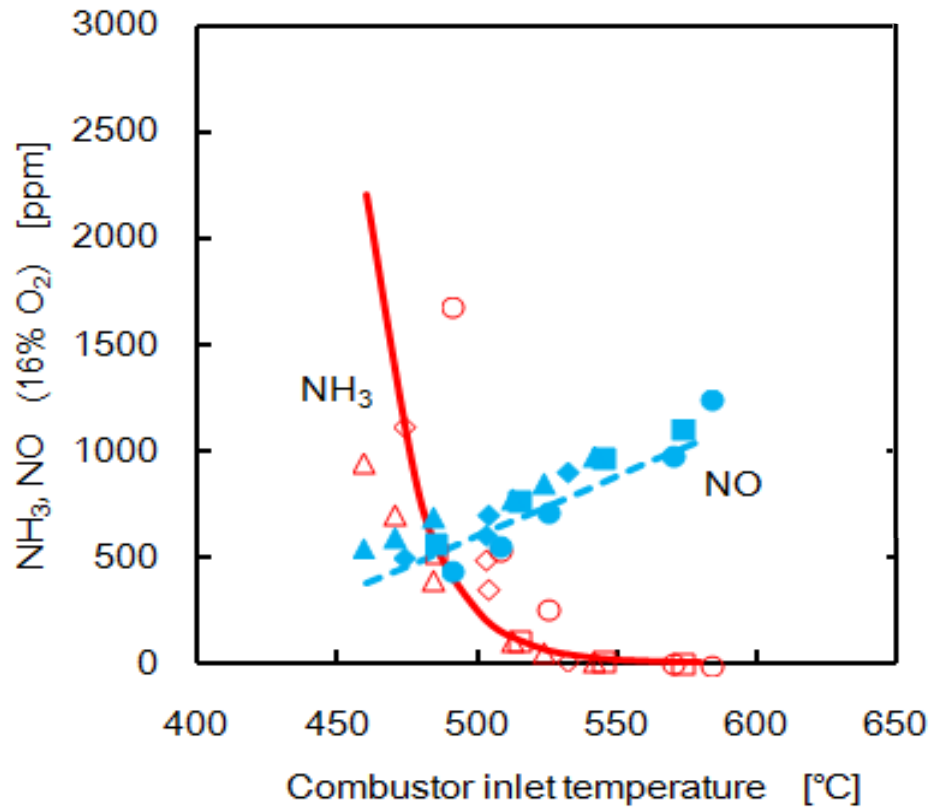
Pure ammonia swirling flame

## ACTUAL FLOW FIELD IN THE COMBUSTOR



Swirling flows generate recirculating flows that recirculate heat and active radicals in the combustor, enhancing the stability of the flame.





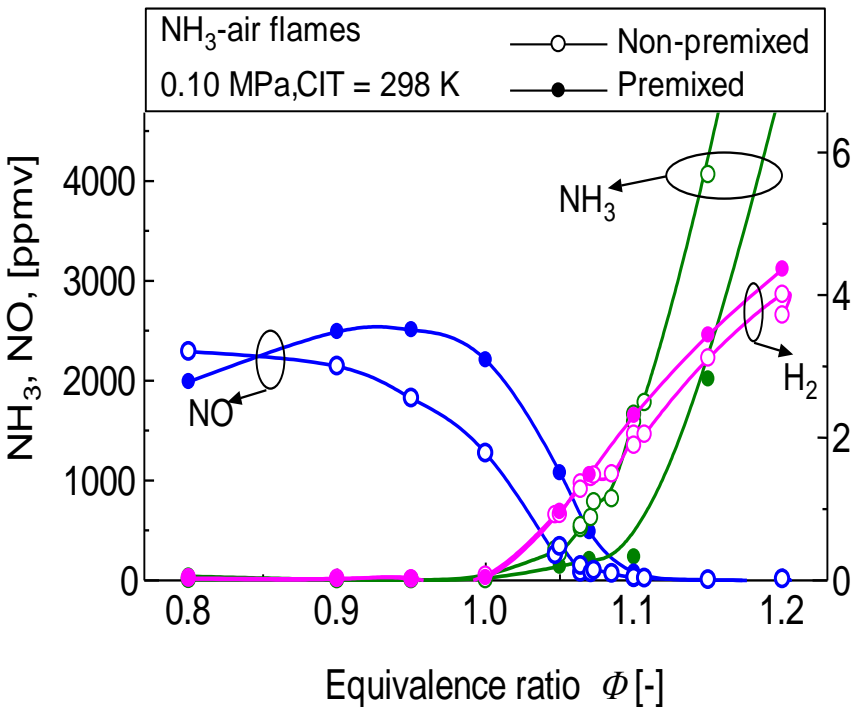
- NO emission was high
- Combustion efficiency was low for CIT less than 500°C

## NO<sub>x</sub> EMISSION TARGET

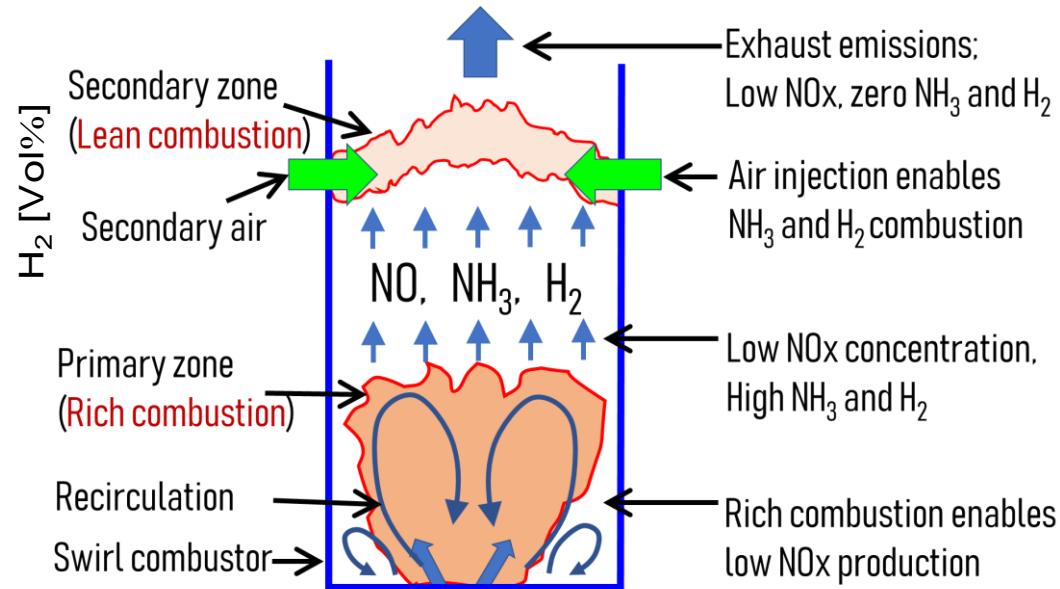
< 200 ppm NO<sub>x</sub>  
from combustor



< 10 ppm NO<sub>x</sub>  
emission



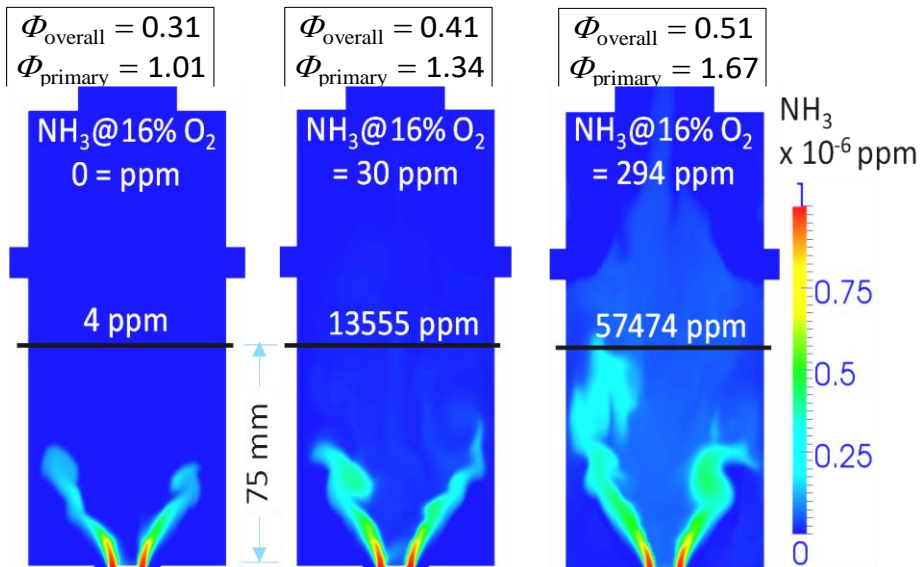
Emissions characteristics of ammonia



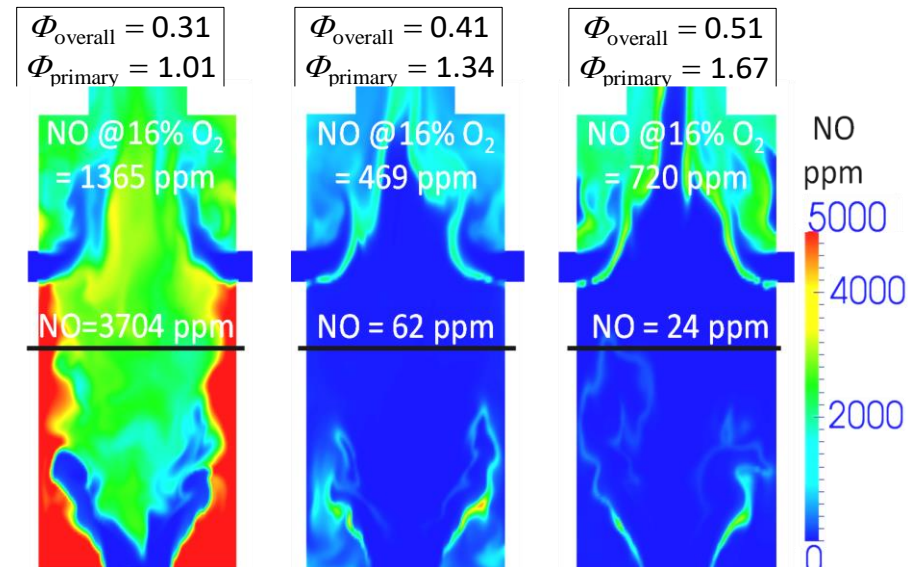
Concept of two-stage combustion

Two-stage rich-lean combustion leverages the low NO<sub>x</sub> production in fuel-rich ammonia flames

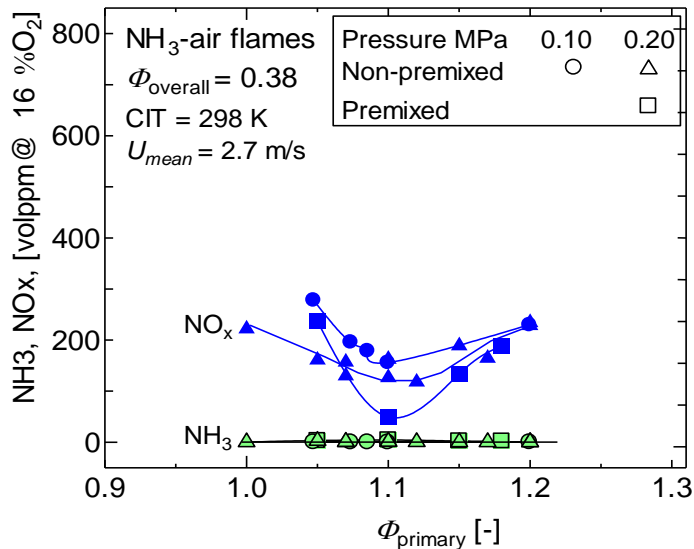




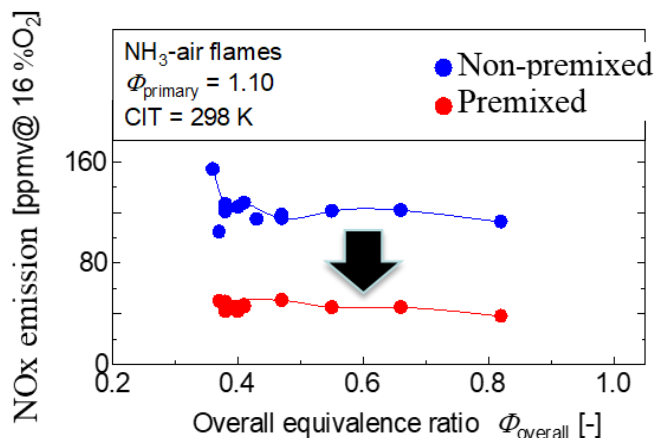
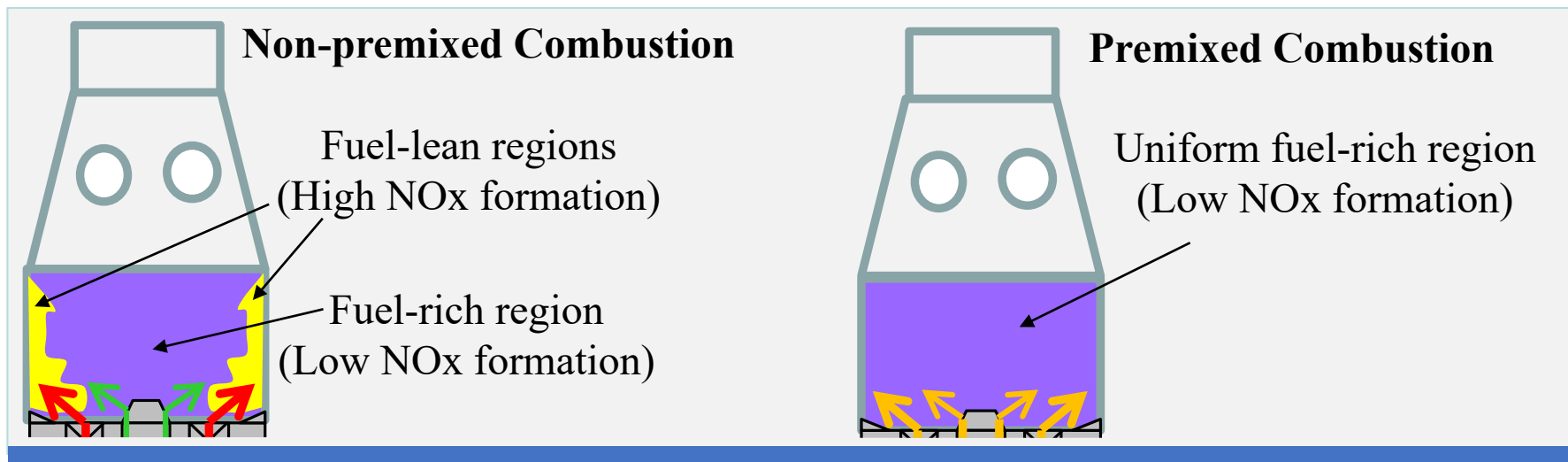
$\text{NH}_3$  profile



$\text{NO}$  profile



A control of the equivalence ratio of the primary zone is necessary for  $\text{NO}_x$  control in two-stage rich-lean combustors. Optimum value  $\cong 1.10$



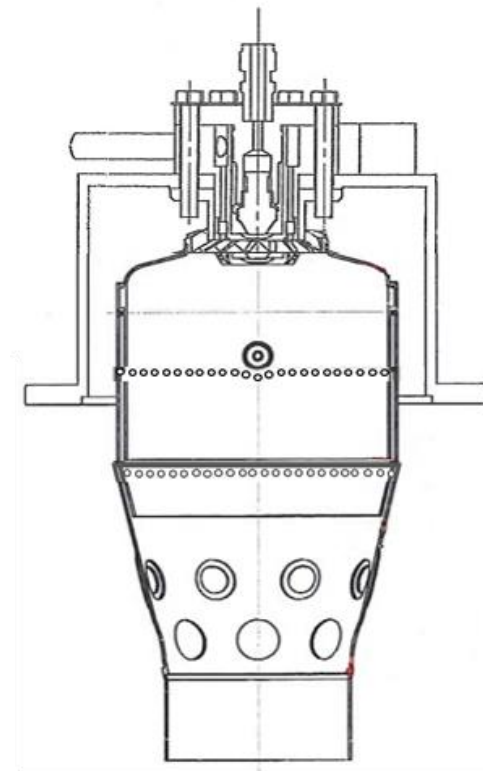
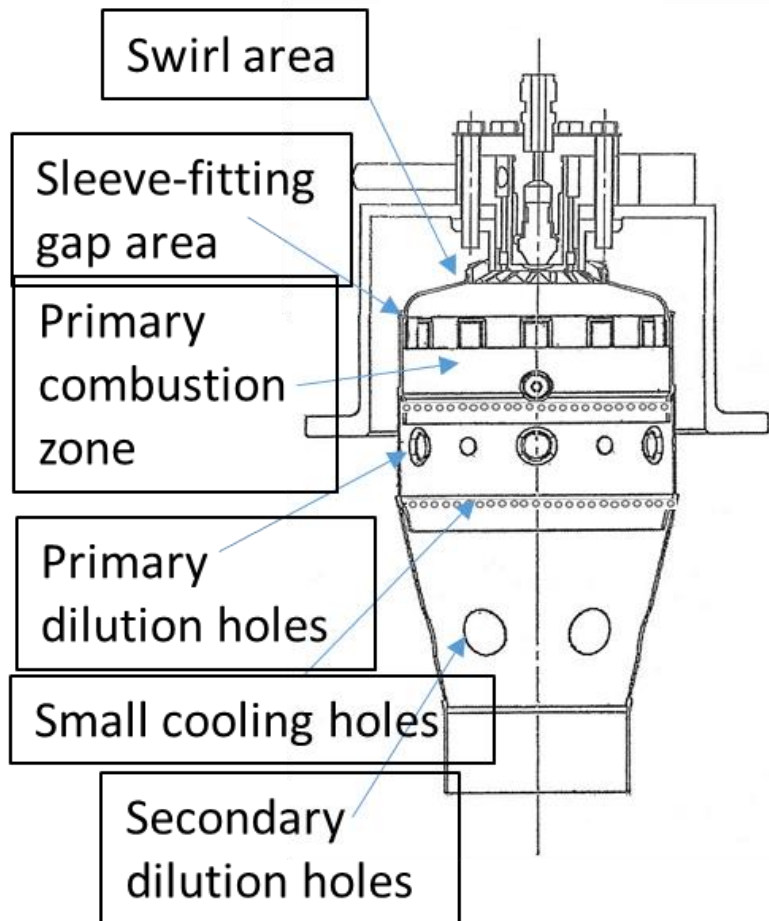
- Non-premixed combustion leads to non-uniform mixture formation.
- Regions of fuel-lean combustion in non-premixed flames produce high amounts of NO<sub>x</sub>.
- Lower NO<sub>x</sub> emissions is recorded with premixed combustion.

The Laboratory studies enabled the design of a low NO<sub>x</sub> combustor

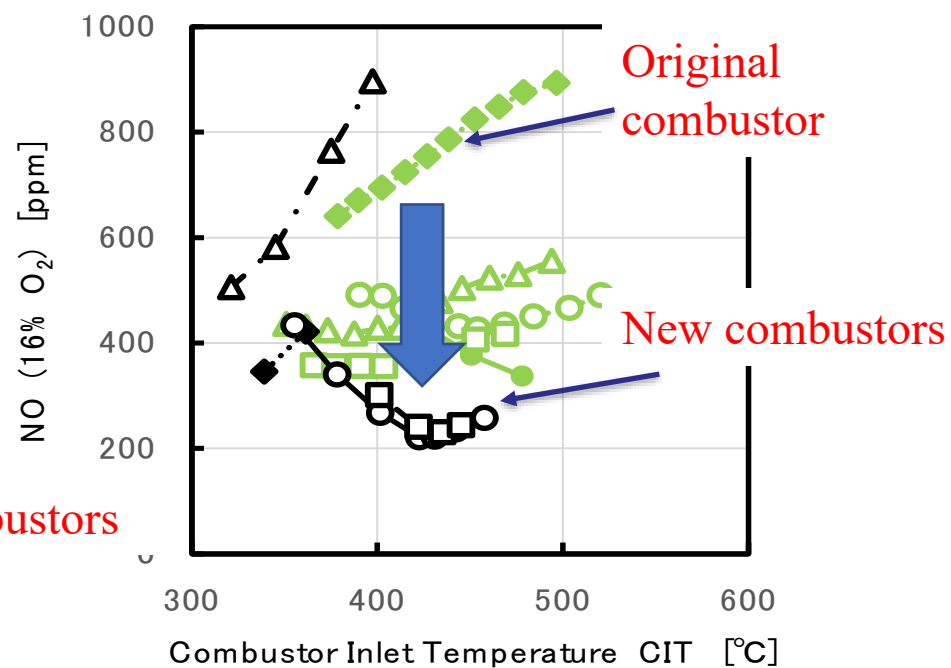
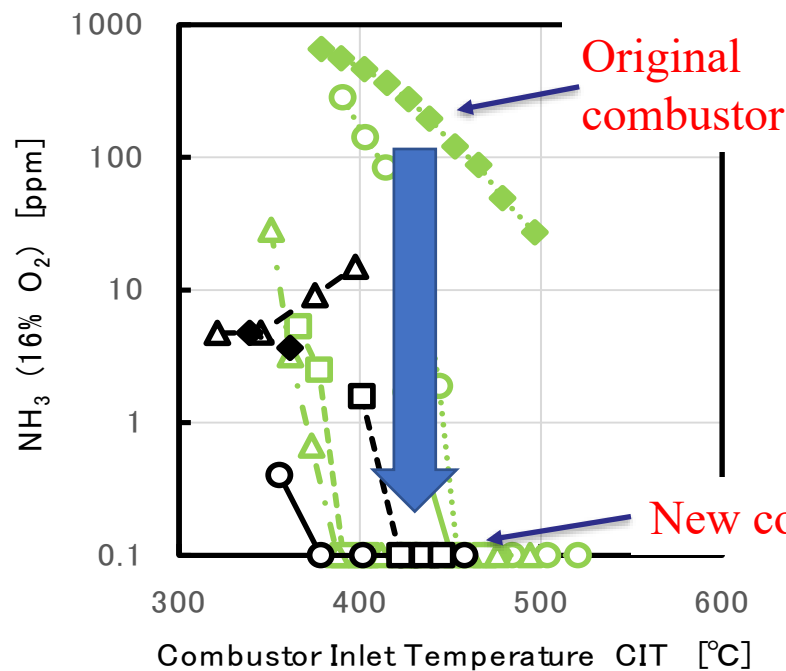
## Original combustor

## Low NO<sub>x</sub> combustor

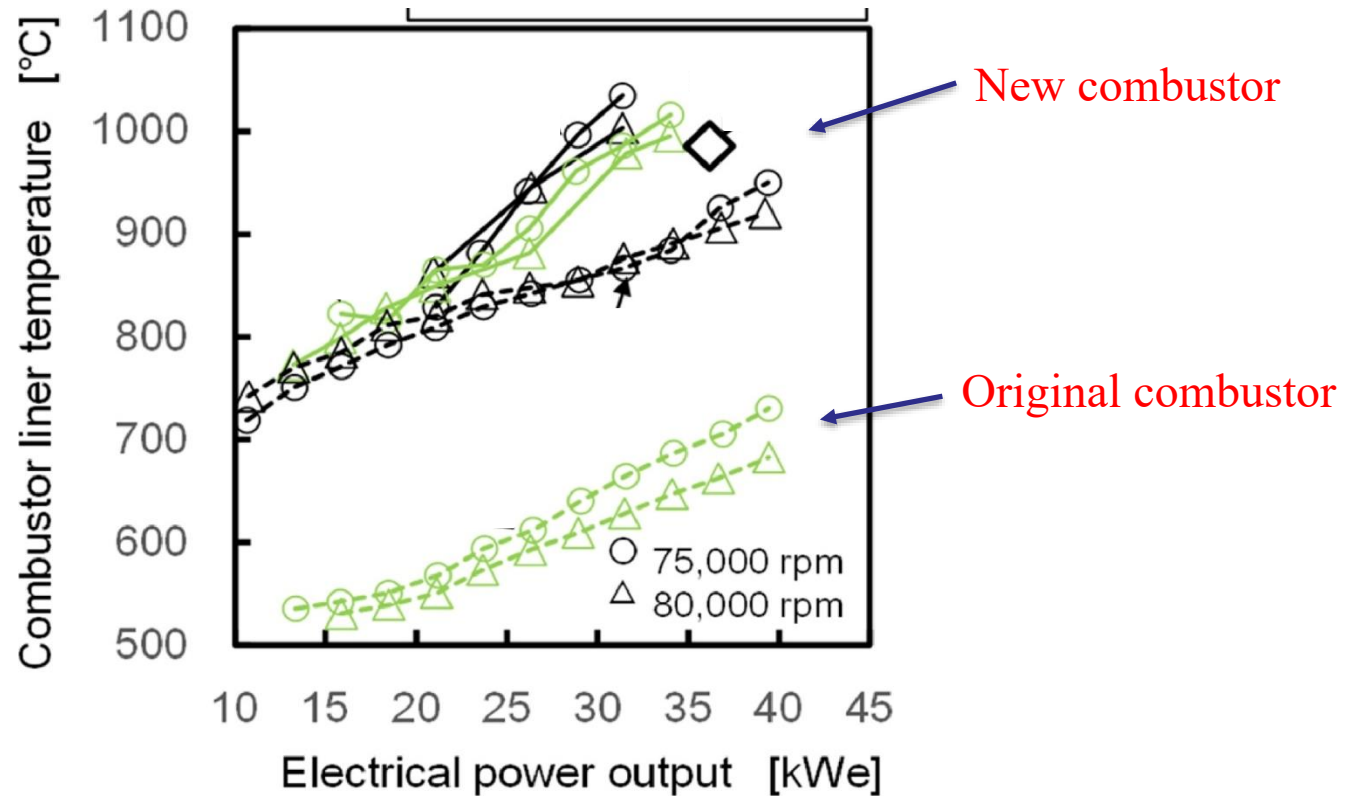
Two-stage rich-lean combustor



Significant improvement in combustion efficiency and reduction in NO emission using two-stage rich-lean combustion.

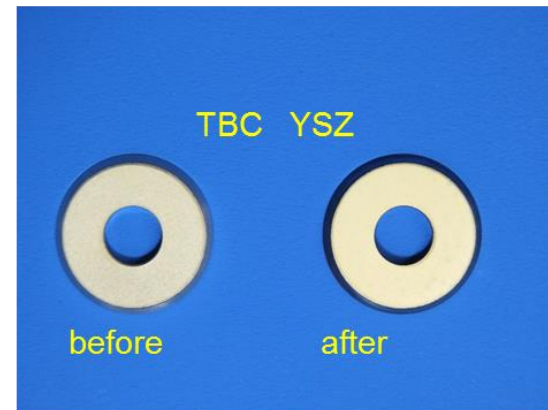
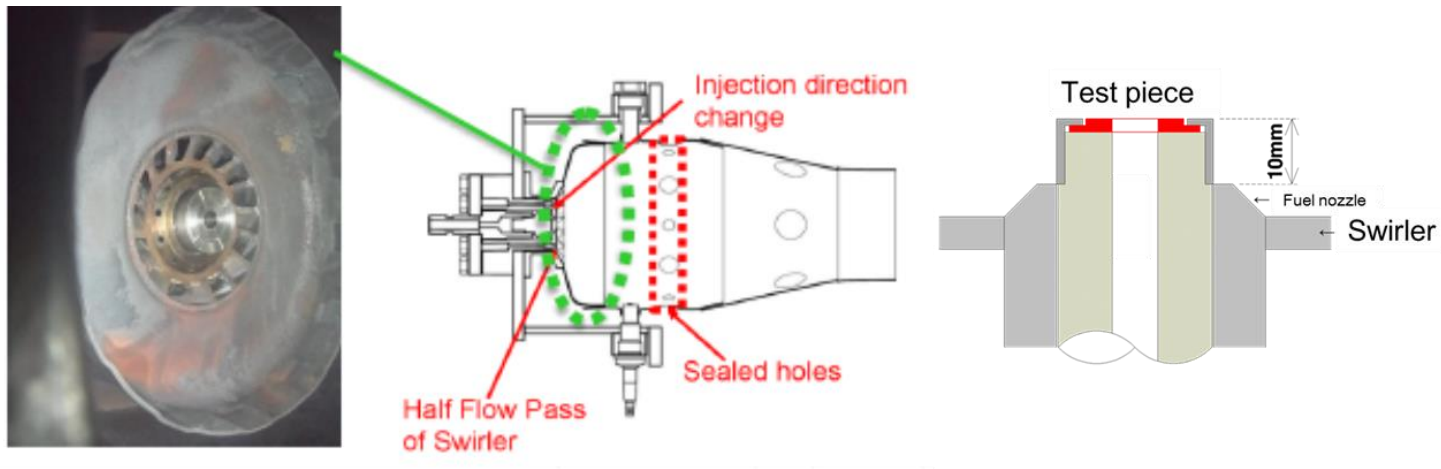


# Combustor wall Temperature



The new combustors resulted in higher combustor liner temperatures

# Temperature resistance testing of materials



Thermal barrier coating using Ytria Stabilized Zirconium powder showed good high temperature resistance for ammonia combustors.

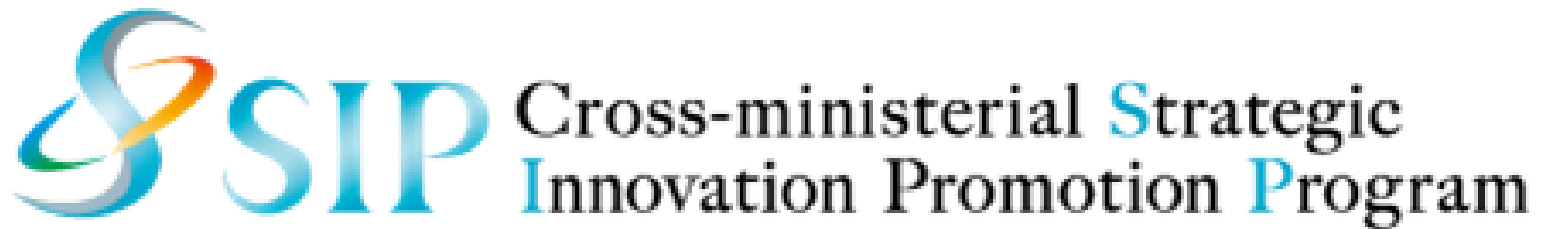
# Summary

- Two-stage rich-lean combustion has been proposed and demonstrated for the control of emissions from ammonia gas turbines.
- NO<sub>x</sub> emission was reduced to 1/3 times of that from the original combustor.
- The new low NO<sub>x</sub> combustor leads to relatively high combustor liner temperatures.
- Temperature resistance of materials was tested under ammonia combustion conditions and thermal barrier coating using Yttria Stabilized Zirconium powder was found more appropriate than Inconel600.



# Acknowledgement

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